

MIN LOAD = 175 lb

MAX LOAD = 225 lb

WORKING DEFLECTION = 0.85 in

ENDS SQUARED AND GROUND

UNPEENED MUSIC WIRE - A228

10% CLASH ALLOWANCE

FORCING FREQUENCY IS 500 rpm

DESIGN FOR INFINITE LIFE

MINIMIZE PACKAGE SIZE

FIND F.O.S. AGAINST FATIGUE, YIELD, AND SURGE

$$k = \frac{225 - 175}{0.85} = 58.82 \text{ lb/in}$$

$$k = \frac{dG}{8C^3N_a}$$

CHOICE OF $C \& d$, $D \& d$, OR $C \& D$ WILL DETERMINE PHYSICAL SIZE OF SPRING. "MINIMIZE PACKAGE SIZE" IS A BIT VAGUE. THE "PACKAGE SIZE" CAN BE CONSIDERED TO BE THE INSTALLED SIZE OF THE SPRING WHICH WOULD BE $[(1+10\%)(0.85\text{in}) + L_s] \times [D+d]$. $C = \frac{D}{d}$, $L_s = d(N_a + 2)$. C MUST BE BETWEEN 4 & 12, AND d MUST BE BETWEEN 0.004 in & 0.312 in

MINIMUM INSTALLED VOLUME IS ABOUT 28 in³ @ $d = 0.25$ in & $C = 9$. (FROM SPREADSHEET) [ASSUMING REASONABLE FOS IN YIELD]

CALCULATE F.O.S. AGAINST YIELD.

$$\tau_{\max} = k_s \frac{8FC}{\pi d^2} = \left(1 + \frac{0.5}{9}\right) \frac{8(225\text{lb})9}{\pi(0.25\text{in})^2} = 87 \text{ kpsi}$$

$$S_{ut} = 225 \text{ kpsi}, S_{ys} = 0.45(225 \text{ kpsi}) = 101,250 \text{ kpsi}$$

$$\text{F.O.S.}_{\text{yield}} = \frac{101}{87} = 1.16$$

CALCULATE F.O.S. IN FATIGUE

$$F_i = 175, F_{max} = 225, F_m = \frac{225 + 175}{2} = 200, F_a = \frac{225 - 175}{2} = 25$$

$$\tau_i = k_s \frac{8 F_i C}{\pi d^2} = \left(1 + \frac{0.5}{9}\right) \frac{8(175)9}{\pi (0.25 \text{ in})^2} = 67.7 \text{ kpsi}$$

$$\tau_m = k_s \frac{8 F_m C}{\pi d^2} = \left(1 + \frac{0.5}{9}\right) \frac{8(200)9}{\pi (0.25 \text{ in})^2} = 77.4 \text{ kpsi}$$

$$\tau_a = k_w \frac{8 F_a C}{\pi d^2} = 1.16 \frac{8(25)9}{\pi (0.25 \text{ in})^2} = 10.7 \text{ kpsi}$$

$$k_w = \frac{4C-1}{4C-4} + \frac{0.615}{C} = \frac{35}{32} + \frac{0.615}{9} = 1.16$$

$$S_{es} = 0.707 \frac{S_{ew} S_{us}}{S_{us} - 0.707 S_{ew}} = 0.707 \frac{45(151)}{151 - 0.707(45)} = 40.3 \text{ kpsi}$$

$$S_{ew} = 45 \text{ kpsi (UNPEELED, EQN 13.12)}$$

$$S_{us} = 0.67(S_{us}) = 0.67(225 \text{ kpsi}) = 151 \text{ kpsi (EQN 13.4)}$$

$$\begin{aligned} FOS_{\text{FATIGUE}} &= \frac{S_{es}(S_{us} - \tau_i)}{S_{es}(\tau_m - \tau_i) + S_{us}\tau_a} = \frac{40.3(151 - 67.7)}{40.3(77.4 - 67.7) + 151(10.7)} \\ &= 1.67 \end{aligned}$$

CALCULATE F.O.S. AGAINST SURGE

$$f = \frac{1}{2} \sqrt{\frac{kg}{w_a}} = \frac{1}{2} \sqrt{\frac{58,82 \text{ lb/in} (386 \text{ m/sec}^2)}{0.841 \text{ lb}}} = 82 \text{ Hz}$$

$$w_a = \frac{\pi^2 d^2 D N_a \gamma}{4} = \frac{\pi^2 (0.25 \text{ in})^2 (2.25 \text{ in}) 8.5 (.285 \text{ lb/in}^3)}{4} = 0.841 \text{ lb}$$

$$FOS_{\text{SURGE}} = \frac{82}{13(8.3)} = \frac{82}{108} = 0.75$$

TECHNICALLY, THE SPRING SHOULD BE REDESIGNED WITH LESS ACTIVE COILS!

IDEALLY, THE SPRING SHOULD BE REDESIGNED WITH LESS OSCILLATING MASS, BUT SINCE THE NATURAL FREQUENCY IS 10X THE FORCING FREQUENCY (13 IS DESIRED), WE'LL LET IT GO.

SPRING DIMENSIONS ARE

$$d = 0.25 \text{ in}$$

$$D = Cd = 9(0.25) = 2.25 \text{ in}$$

$$N_a = \frac{dG}{8C^3k} = \frac{0.25 \text{ in} (11.7 \times 10^6 \text{ PSI})}{8(9)^3 58.82 \text{ lb/in}} = 8.5 \quad (\text{NEAREST } \frac{1}{4})$$

$$L_s = d(N_a + 2) = 0.25(8.5 + 2) = 2.625 \text{ in}$$

$$y_c = 0.085 \text{ in}$$

$$y_w = 0.85 \text{ in}$$

$$y_i = \frac{175 \text{ lb}}{58.82 \text{ lb/in}} = 2.975 \text{ in}$$

$$L_f = L_s + y_c + y_w + y_i = 6.54 \text{ in}$$

$$\text{installed length} = L_s + y_c + y_w = 3.56 \text{ in}$$