

BRAKES ~ EXAMPLE PROBLEM

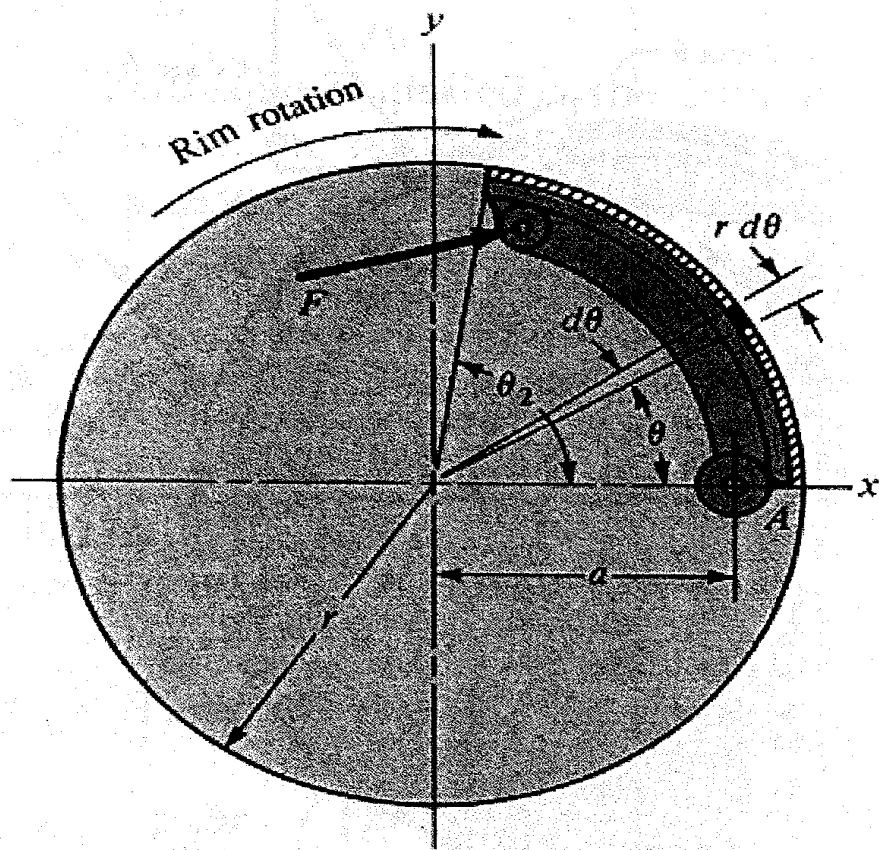
Reference: Shigley, 5th Edition

Problem 16.4

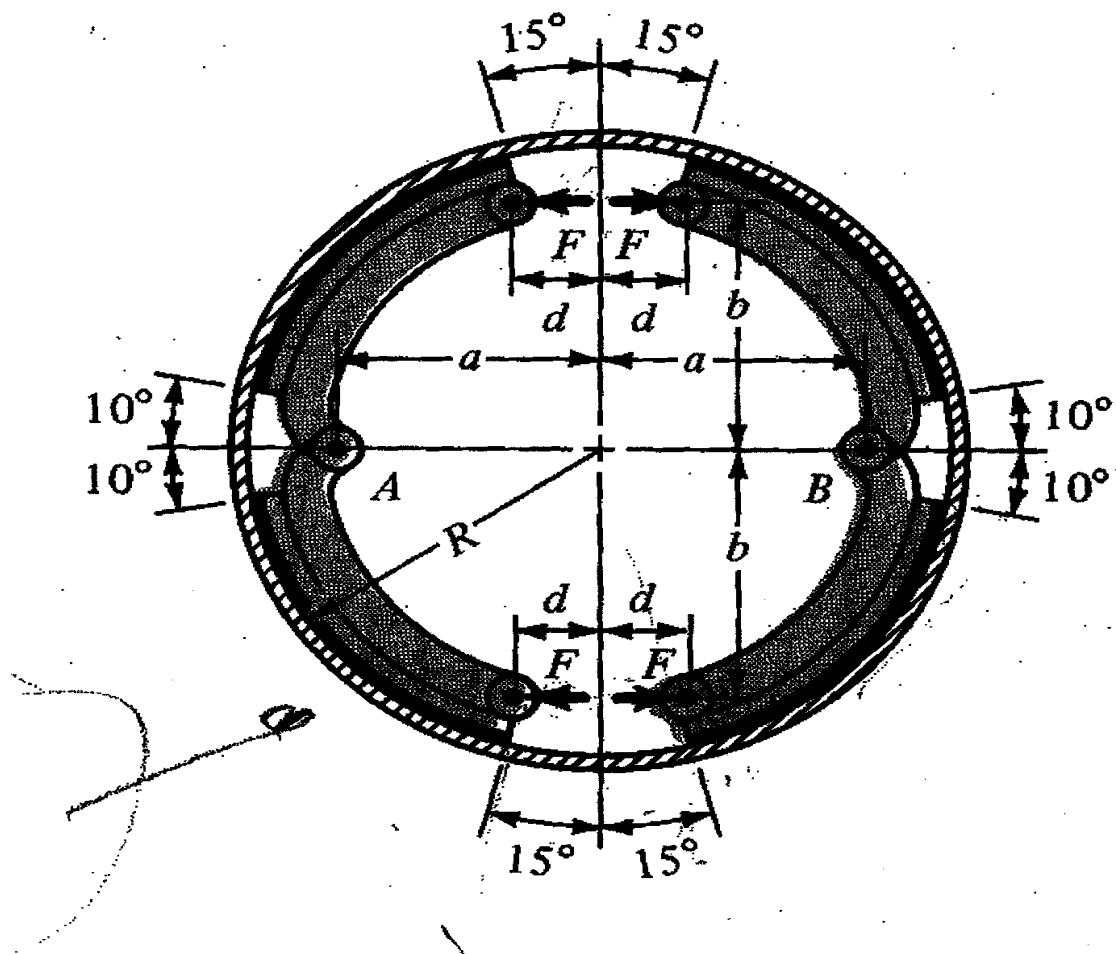
The figure below shows a 400-mm diameter brake drum with four (4) internally expanding shoes. Each of the hinge pins A and B supports a pair of shoes. The actuating mechanism is to be arranged to produce the same force F on each shoe. The face width of the shoes is 75-mm. The material used has a coefficient of friction of 0.24 and a maximum pressure of 1-MPa. Determine:

- (a) Actuating force
- (b) Braking capacity
- (c) Noting the rotation may be in either direction, compute the hinge-pin reactions.

$$p_{\theta} = p_a \frac{\sin \theta}{\sin \theta_a}$$



Internal Friction Shoe



Dimensions are in millimeters and are:

$$a = 150$$

$$b = 165$$

$$d = 50$$

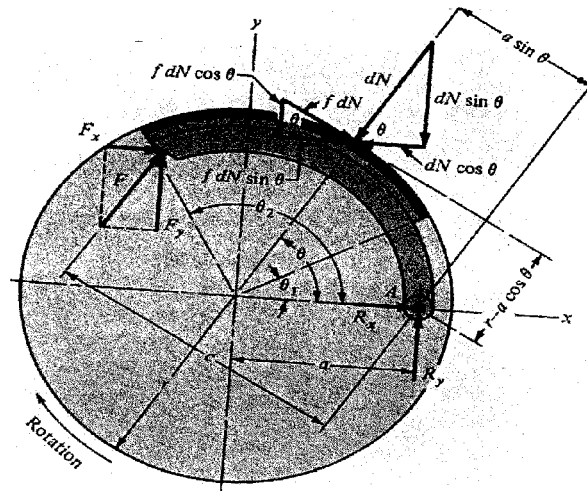
p = unit pressure acting upon an element of area of the frictional material located at an angle θ from the hinge pin.

p_a = maximum pressure located at an angle θ_a from the hinge pin.

$p = \max @ \theta = 90\text{-degrees}$.

If the *toe angle*, θ_2 , is less than 90-degrees, maximum pressure will occur at the toe.

When $\theta = 0$, $p = 0$; material at the toe contributes little to the friction force.



Brake Shoe Forces

$$M_S = \frac{f p_a b r}{\sin \theta_a} \int_{\theta_1}^{\theta_2} \sin(\theta) (r - a \cos \theta) d\theta$$

= moment of frictional forces about pin.

$$M_N = \frac{p_a b r a}{\sin \theta_a} \int_{\theta_1}^{\theta_2} \sin^2 \theta d\theta$$

= moment of normal forces about pin

$$F = \frac{M_N - M_S}{C}$$

~ Actuating Force

$$M_N = M_S$$

~ self-locking

a must be s.t. $M_N > M_S$ - for self-energizing.

$$M_W = \int dN (a \sin \theta) d\theta$$

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$$T = \int S_r dA = \frac{S_p b r^2}{\sin \theta_a} \int_{\theta_i}^{\theta_r} \sin \theta d\theta$$

$$T = \frac{2\pi a b r^2 (\cos \theta_1 - \cos \theta_2)}{\sin \theta_a}$$

$$R_x = \int dN \cos \theta - \int \frac{p_{cr}}{\sin \theta_a} \int_{\theta_1}^{\theta_2} \sin \theta \cos \theta d\theta - F_x$$

$$R_x = \int dN \cos \theta - \int \frac{p_{cr}}{\sin \theta_a} \int_{\theta_1}^{\theta_2} \sin \theta \cos \theta d\theta - F_x$$

$$R_y = \frac{P_a b r}{\sin \theta_a} \int_{\theta_1}^{\theta_2} \sin^2 \theta d\theta + \int_{\theta_1}^{\theta_2} \sin \theta \cos \theta d\theta$$

$$R_y = \frac{P_a b r}{\sin \theta_a} \int_{\theta_1}^{\theta_2} \sin^2 \theta d\theta + \int_{\theta_1}^{\theta_2} \sin \theta \cos \theta d\theta$$

$$F = \frac{M_A + M_B}{c}$$

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$$A = \int_{\theta_1}^{\theta_2} \sin \theta \cos \theta d\theta = \frac{1}{2} \sin^2 \theta \Big|_{\theta_1}^{\theta_2}$$

$$B = \int_{\theta_1}^{\theta_2} \sin^2 \theta d\theta = \left[\frac{\theta}{2} - \frac{\sin 2\theta}{4} \right] \Big|_{\theta_1}^{\theta_2}$$

$$\rightarrow R_x = \frac{p_a b r}{\sin \theta_a} (A - \int B) - F_x$$

$$R_y = \frac{p_a b r}{\sin \theta_a} (B + \int A) - F_y$$

$$\leftarrow R_x = \frac{p_a b r}{\sin \theta_a} (A - \int B) - F_x$$

$$R_y = \frac{p_a b r}{\sin \theta_a} (B - \int A) - F_y$$

Reference system ~ origin at center drum.

- Positive x-axis, thru hinge pin
- Positive y-axis, in direction of shoe.

Problem:

$$\theta_1 = 10^\circ$$

$$\theta_2 = 75^\circ$$

$$\therefore \theta_a = 75^\circ$$

~

$$\text{Let } \alpha = \left[r \int_{\theta_1}^{\theta_2} \sin \theta d\theta - a \int_{\theta_1}^{\theta_2} \sin \theta \cos \theta d\theta \right]$$

$$\textcircled{1} \quad \alpha = \left[r \int_{\theta_1}^{\theta_2} \sin \theta d\theta - a A \right]$$

$$\textcircled{c} B = \int_{\theta_1}^{\theta_2} \sin^2 \theta d\theta = \left[\frac{\theta}{2} - \frac{\sin 2\theta}{4} \right]_{10}^{75}$$

$$A = \int_0^{\pi/2} \int_0^{2\pi} \sin \theta \cos \theta d\theta d\phi = \frac{1}{2} \sin^2 \theta \Big|_0^{\pi/2} \int_0^{2\pi} d\phi$$

$$A = 0.45 \bar{1}$$

$$B = 0.527$$

$$\alpha = r[-\cos\theta]_{\theta_1}^{\theta_2} - aA$$

$r = 200 \text{ mm}$

$$a = 150 \text{ mm}$$

$$\alpha = 77.5 \text{ mm}$$

$$M_s = \left(\frac{S_p a b r}{\sin \theta_a} \right) \frac{N}{m^2}$$

$$= \frac{(0.24) [1 \times 10^6] (0.675 \text{ m}) (0.200 \text{ m})}{\sin(75^\circ)} \quad \downarrow 0.0775 \text{ m}$$

$$(0.33) \text{ m}$$

$$M_s = 290 \text{ N}\cdot\text{m}$$

$$M_N = \frac{p_{abra}}{\sin \theta_a} B$$

$$M_N = \frac{(1 \times 10^6 \frac{N}{m^2})(0.075m)(.2m)(.15m)}{\sin(75)} (0.53)$$

$$M_H = 1230 \text{ N}\cdot\text{m}$$

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$$F = \frac{M_N - M_S}{c}$$

$$= \frac{1230 - 289}{.165} \frac{\text{N} \cdot \text{m}}{\text{m}}$$

$$F = \begin{cases} 5.7 \text{ KN} \\ 5700 \text{ N} \end{cases}$$

For the primary shoe:

$$T = \frac{S_p a b^2 (\cos \theta_1 - \cos \theta_2)}{\sin \theta_a}$$

$$T = \frac{(.15 \text{ m})(1 \times 10^6 \frac{\text{N}}{\text{m}^2})(\cos 75^\circ - \cos 10^\circ)(0.24)(.2)^2 \text{ m}^2}{\sin 75^\circ}$$

$$T = 541 \text{ N} \cdot \text{m}$$

For the secondary shoe:

$$F = \frac{M_N + M_S}{c} = 5700 \text{ N}$$

$$(F_{\text{primary}} = F_{\text{secondary}})$$

$$M_N = \frac{1230}{10^6} \text{ Pa}$$

$$M_S = \frac{289}{10^6} \text{ Pa}$$

$$5700 = \frac{(1230/10^6) \text{ Pa} + (289/10^6) \text{ Pa}}{.165}$$

$$p_a = 0.62 \times 10^6 \text{ Pa}$$

$$T = 335 \text{ N}\cdot\text{m}$$

$$\begin{aligned} T_{\text{TOTAL}} &= T_L + T_R \\ &= (2)(911) + (2)(335) \end{aligned}$$

$$\begin{aligned} T_{\text{TOTAL}} &= 1750 \text{ N}\cdot\text{m} \\ &= \text{Braking capacity} \end{aligned}$$

$$R_x = -0.65 \text{ kN}$$

$$R_y = 3.87 \text{ kN}$$

Primary

$$R_x = -0.14 \text{ kN}$$

$$R_y = 4.02 \text{ kN}$$

Secondary